

SVINGNINGER

[YF 14; LL 9]

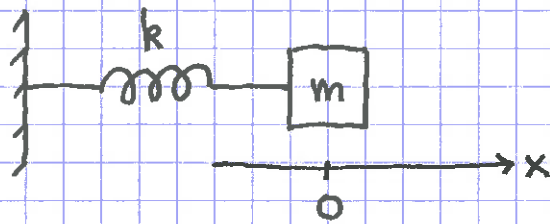
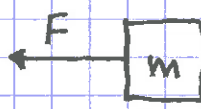
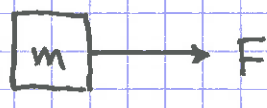
= oscillasjoner

= periodisk oppførsel omkring likevekt

Eks: masse/fjær, pendel, gitarstreng, luft i orgelpipe, atomer i molekyler og krystaller

Harmonisk oscillator

[YF 14.2; LL 9.1-9.3]

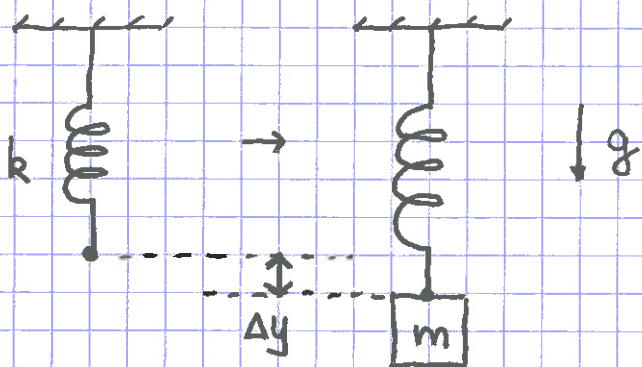
Likevekt ($F=0$) med m i $x=0$ (evt. CM i $x=0$)Strukket fjær, $x > 0$: Sammenpresset fjær, $x < 0$: Hookes lov (ideell fjær): $|F| \sim |x|$

$$\Rightarrow \boxed{\vec{F} = -k \times \hat{x}}$$

Fjærkonstanten: k

$$[k] = \text{N/m}$$

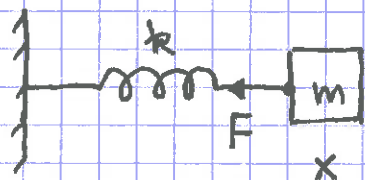
Vertikalt i tyngdefeltet:



Likevekt:

$$k \cdot \Delta y = mg$$

$$\Rightarrow \Delta y = mg/k$$



$$N2: -kx = m\ddot{x}$$

$$\Rightarrow \ddot{x} + \frac{k}{m}x = 0$$

Innfør $\omega_0 = \sqrt{k/m}$

$\Rightarrow$

$$\ddot{x} + \omega_0^2 x = 0$$

Beregelseslikning
for enkel harmonisk
oscillator, i en dimensjon

Løsning [TMA4110, øving 3]: Frie svingn. uten damping.

$$x(t) = A \cos(\omega_0 t + \varphi) \quad \text{evt.} \quad x(t) = B \cos \omega_0 t + C \sin \omega_0 t$$

der A og φ , evt. B og C , fastlegges via

2 initialbetingelser, f.eks. $x(0) = x_0$ og $\dot{x}(0) = v_0$

[Bruk $\cos(a+b) = \cos a \cos b - \sin a \sin b$ til å
finne sammenhengen mellom A, φ og B, C !]

$$x(t) = A \cos(\omega_0 t + \varphi)$$

A = amplitude = max utsving fra likevekt; $[A] = [x]$

ω_0 = vinkelfrekvens; $[\omega_0] = s^{-1}$

$T = 2\pi/\omega_0$ = periode = tid pr svingning; $[T] = s$

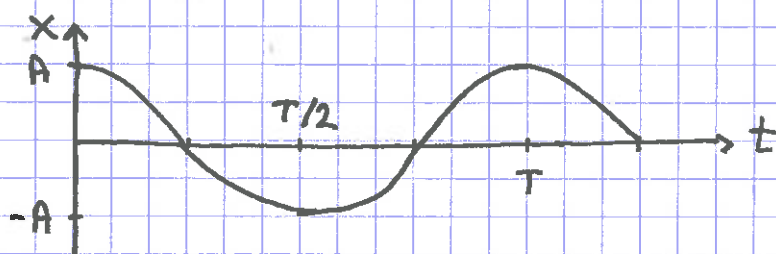
$f = 1/T$ = frekvens = antall svingninger pr tidsenhet; $[f] = \text{Hz} = s^{-1}$

$\omega_0 t + \varphi$ = svingningens fase

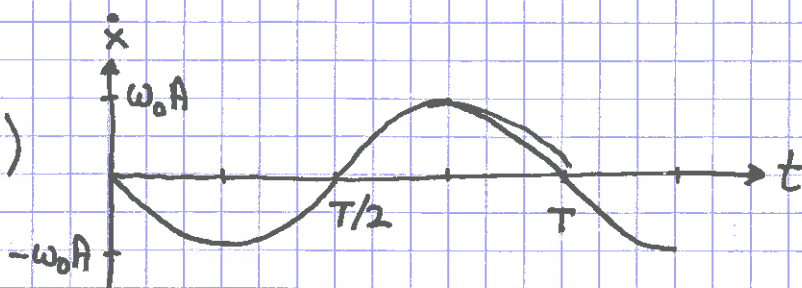
φ = fasekonstant; $[\varphi] = 1$

Anta $\varphi = 0$, $A > 0$:

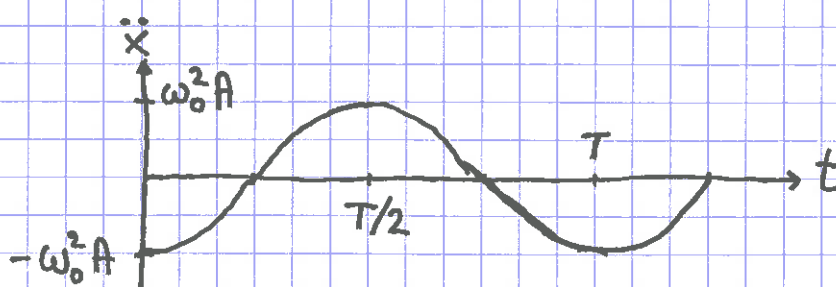
$$x(t) = A \cos \omega_0 t$$



$$\begin{aligned} \dot{x}(t) &= -\omega_0 A \sin \omega_0 t \\ &= \omega_0 A \cos(\omega_0 t + \frac{\pi}{2}) \end{aligned}$$



$$\begin{aligned} \ddot{x}(t) &= -\omega_0^2 A \cos \omega_0 t \\ &= \omega_0^2 A \cos(\omega_0 t + \pi) \end{aligned}$$



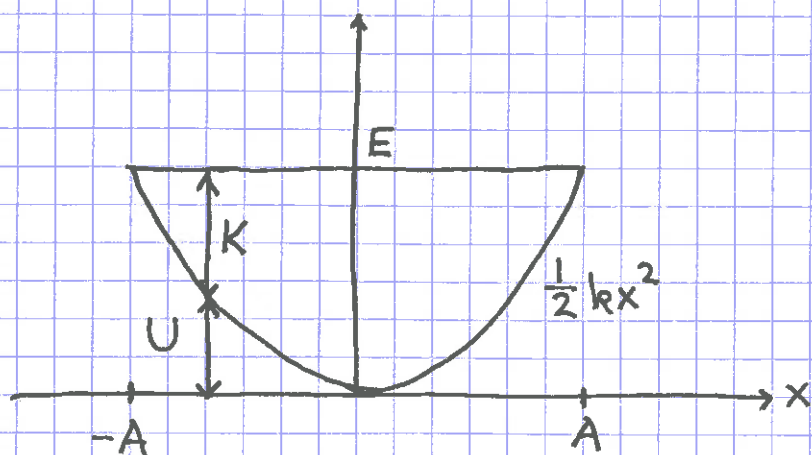
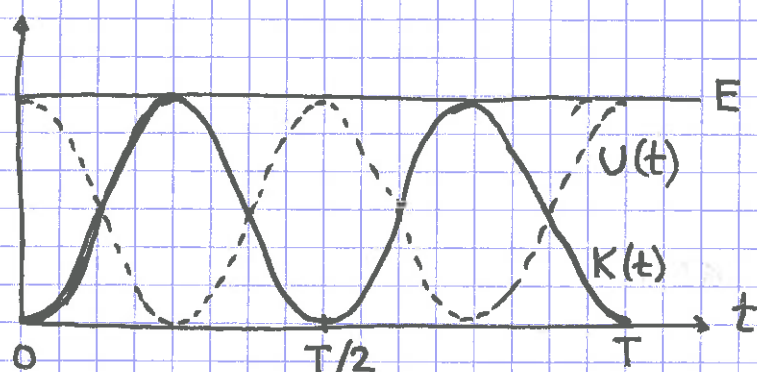
Energi i harmonisk oscillator [YF 14.3; LL 9.4] (59)

$$K(t) = \frac{1}{2} m \dot{x}(t)^2 = \frac{1}{2} m \omega_0^2 A^2 \sin^2 \omega_0 t = \frac{1}{2} k A^2 \sin^2 \omega_0 t$$

$$U(t) = - \int_0^x F[x(t)] dx = - \int_0^x [-kx(t)] dx = \frac{1}{2} k x(t)^2 \\ = \frac{1}{2} k A^2 \cos^2 \omega_0 t$$

$$\Rightarrow E = K + U = \frac{1}{2} k A^2 = \frac{1}{2} m \omega_0^2 A^2 = \text{konstant}$$

Dvs: Konservativt system (mek. energi E er bevart)

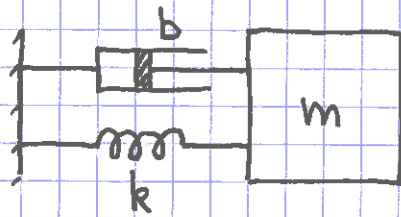


Dempet fri svingning

[YF 14.7; LL 9.7]

(60)

Anta friksjon som for langsom bevegelse i fluid (s.12):



$$f = -b\dot{x}$$

Netto kraft på m: $-kx - b\dot{x}$

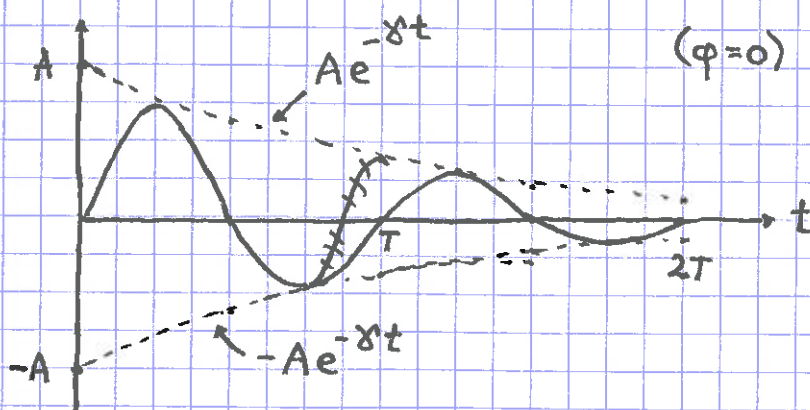
$$N2: -kx - b\dot{x} = m\ddot{x}$$

$$\Rightarrow \ddot{x} + 2\gamma\dot{x} + \omega_0^2 x = 0 ; \quad \gamma = \frac{b}{2m}, \quad \omega_0^2 = \frac{k}{m}$$
$$[\gamma] = [\omega_0] = s^{-1}$$

Løsning [TMA 4110, øving 3]:

Underkritisk demping, $\gamma < \omega_0$ ($b < 2\sqrt{k \cdot m}$)

$$x(t) = A e^{-\gamma t} \sin(\omega t + \varphi) ; \quad \omega = \sqrt{\omega_0^2 - \gamma^2} < \omega_0$$

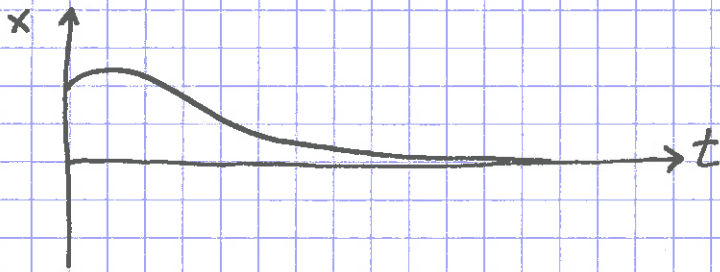


Amplituden $A e^{-\gamma t}$ avtar eksponentielt med t

Overkritisk demping, $\gamma > \omega_0$

$$x(t) = A e^{-\alpha_1 t} + B e^{-\alpha_2 t}$$

$$\alpha_1 = \gamma + \sqrt{\gamma^2 - \omega_0^2}, \quad \alpha_2 = \gamma - \sqrt{\gamma^2 - \omega_0^2}$$

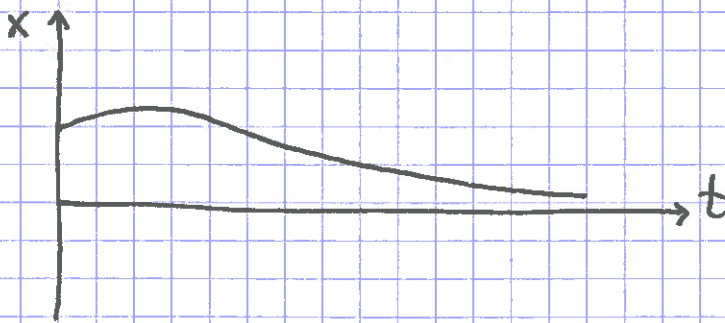


Ingen svingninger

Kritisk demping, $\gamma = \omega_0$

$$x(t) = A e^{-\gamma t} + B t e^{-\gamma t}$$

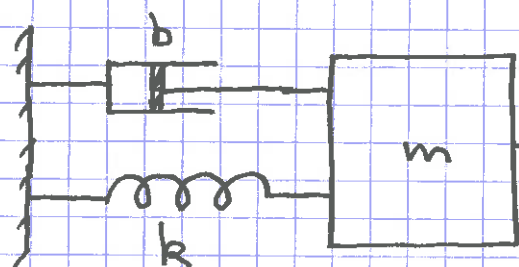
("Dobbel rot",
 $\alpha_1 = \alpha_2 = \gamma$)



Eks: Støtdempere i bil.

Mest behagelig på humpete veier.

02.10.14



$$F_y(t) = F_0 \cos \omega t$$

= ytre kraft, antas harmonisk

$$N2: -kx - b\dot{x} + F_0 \cos \omega t = m\ddot{x}$$

$$\Rightarrow \ddot{x} + 2\gamma \dot{x} + \omega_0^2 x = \frac{F_0}{m} \cos \omega t$$

Løsning:

$$x(t) = x_h(t) + x_p(t)$$

der homogen løsn. x_h oppfyller $\ddot{x}_h + 2\gamma \dot{x}_h + \omega_0^2 x_h = 0$

og partikulærløsn. x_p oppfyller $\ddot{x}_p + 2\gamma \dot{x}_p + \omega_0^2 x_p = \frac{F_0}{m} \cos \omega t$

I starten (før $e^{-\gamma t} \ll 1$) bidrar både x_h og x_p til innsvingningsforløpet.

Etter hvert blir $\gamma t \gg 1$, $e^{-\gamma t} \approx 0$ og $x_h(t) \rightarrow 0$, slik at $x(t) = x_p(t)$.

$$\text{Gjetter: } x_p(t) = A(\omega) \sin(\omega t + \varphi)$$

Innsetting av x_p , $\dot{x}_p$ og $\ddot{x}_p$ resulterer i:

$$A(\omega) = \frac{F_0/m}{\sqrt{(\omega^2 - \omega_0^2)^2 + (2\gamma\omega)^2}}; \quad \varphi(\omega) = \arctan\left(\frac{\omega_0^2 - \omega^2}{2\gamma\omega}\right)$$

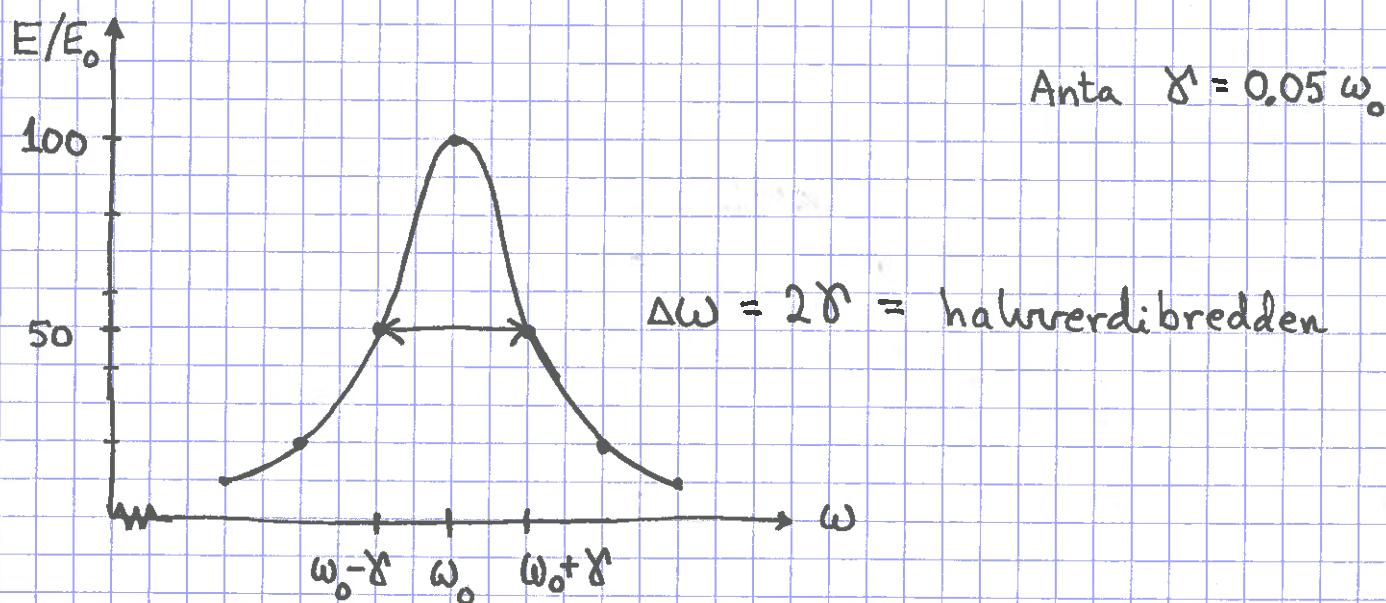
(med $\omega_0^2 = k/m$, $2\gamma = b/m$)

Resonans: Anta svak damping, $\gamma \ll \omega_0$. Ser da (63) at amplituden A blir stor hvis ytre kraft har frekvens $\omega = \omega_0 = \sqrt{k/m}$ = systemets egenfrekvens (evt. resonansfrekvens): $A \rightarrow \infty$ når $\gamma \rightarrow 0$ og $\omega \rightarrow \omega_0$.

La oss plote oscillatorens energi (se s. 59):

$$E = \frac{1}{2} k A^2 = \frac{1}{2} k \frac{F_0^2}{m^2} \cdot \frac{1}{\omega_0^4} \cdot \frac{\omega_0^4}{(\omega^2 - \omega_0^2)^2 + (2\gamma\omega)^2}$$

$$= \underbrace{\frac{F_0^2}{2k}}_{E_0} \cdot \underbrace{\frac{\omega_0^4}{(\omega^2 - \omega_0^2)^2 + (2\gamma\omega)^2}}_{\text{dimensjonsløs størrelse}} \quad [\text{OK for } \omega \text{ nær } \omega_0]$$



Q-faktor: Mål for resonanstoppens grad av "skarphet"

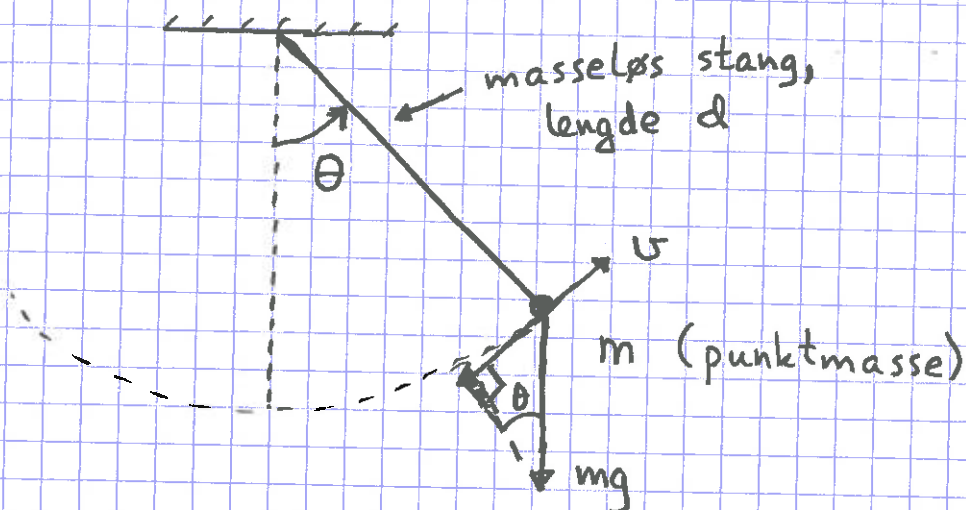
$$Q = \frac{\omega_0}{\Delta\omega} = \frac{\omega_0}{2\gamma}$$

Jo mindre damping, jo større verdi for Q .

(Q for "quality"!) Her: $Q = 10$

Matematisk pendel [YF 14.5; LL 9.6]

(64)



N2 || sirkelbuen:

$$-mg \sin \theta = m a_{||} = m \frac{dv}{dt} = m l \frac{d\omega}{dt} = m l \frac{d^2\theta}{dt^2}$$

$$\Rightarrow \ddot{\theta} + \frac{g}{l} \sin \theta = 0$$

Hvis vi hele tiden har små udsving, $|\theta| \ll 1$,
er $\sin \theta \approx \theta$

$$\Rightarrow \ddot{\theta} + \omega_0^2 \theta = 0 \quad ; \quad \omega_0 = \sqrt{g/l}$$

$$\Rightarrow \theta(t) = \theta_0 \cos \omega_0 t$$

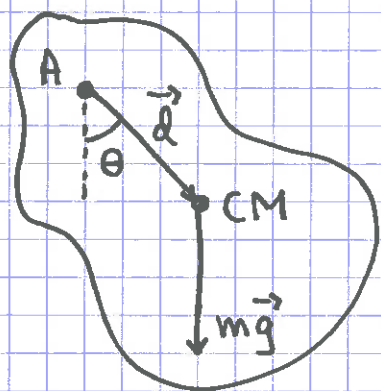
Dvs enkel harmonisk oscillator, med
svingeperiode $T = 2\pi/\omega_0 = 2\pi\sqrt{l/g}$

[Øving 7: Numerisk løsning av $\ddot{\theta} + \frac{g}{l} \sin \theta = 0$. Obligatorisk!]

Fysisk pendel

[YF 14.6; LL 9.6]

(65)



Stivt legeme, masse m , svinger om akse gjennom A, treghetsmoment I mhp rot.aksen, $\vec{d} = \vec{R}_{CM}$ med A som ref.punkt

N2 for rotasjon om fast akse gjennom A:

$$\tau = I\dot{\omega} = I\ddot{\theta} \quad \text{med} \quad \tau = -d \cdot mg \cdot \sin\theta$$

$$\Rightarrow \ddot{\theta} + \frac{mgd}{I} \sin\theta = 0$$

Hvis små utsving: $\sin\theta \approx \theta$

$$\Rightarrow \ddot{\theta} + \omega_0^2 \theta = 0 \quad ; \quad \omega_0 = \sqrt{\frac{mgd}{I}}$$

[Sjekk for punktmasse m i CM, masseløs stang med lengde d : $I = m \cdot d^2 \Rightarrow \omega_0 = \sqrt{g/d}$, som s. 64, OK!]

Steiners sats: $I = I_0 + md^2$; I_0 er tregh.mom mhp akse gjennom CM

$$\Rightarrow T = 2\pi/\omega_0 = 2\pi \sqrt{\frac{I}{mgd}} = 2\pi \sqrt{\frac{I_0 + md^2}{mgd}}$$

[$\Rightarrow T$ blir stor når $d \rightarrow 0$, som ventet]